Simulating Two-Phase Viscoelastic Flows Using Moving Finite Element Methods

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Abstract. Phase-field models provide a way to model fluid interfaces as having finite thickness; the interface between two immiscible fluids is treated as a thin mixing layer across which physical properties vary steeply but continuously. One of the main challenges of this approach is in resolving the sharp gradients at the interface. In this paper, moving finite-element methods are used to simulate interfacial dynamics of two-phase viscoelastic flows. The finite-element scheme can easily accommodates complex flow geometry and the moving mesh strategy can cluster more grid points near the thin interfacial areas where the solutions have large gradients. A diffused monitor function is used to ensure high quality meshes near the interface. Several numerical experiments are carried out to demonstrate the effectiveness of the moving mesh strategy.

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Key words: Moving finite-element method, two-phase flow, viscoelastic flow, non-Newtonian flow.

1 Introduction

Modeling and simulating two-phase viscoelastic flows have been challenging both mathematically and technically. There have been many computational techniques developed to tackle the problem, including diffuse-interface methods [1], interface tracking methods [17], level-set methods [16, 20], finite-element methods with adaptive mesh refinements [22] and spectral methods with adaptive mesh redistribution [7]. The governing...
equations require additional constitutive equations for stress tensor and numerical computations require higher mesh resolution around fluid interfaces. Unlike the Newtonian flow whose local stress is proportional to the local strain rate, for polymer fluids the local stress usually depends on its deformation history due to the long chain molecular structure. Therefore an additional constitutive relation between the stress and the strain of flow such as Upper Convected Maxwell (UCM) model by Oldroyd [13] should be coupled into the flow system.

When comes to multi-phase flows, problems arise due to the sharp interface between fluids. The classical jump conditions and surface tension are introduced into the Navier-Stokes equations in which the interface has zero width. Different approaches such as volume-of-fluid (VOF) and level-set method have been developed to handle this problem, see, e.g., [16, 20]. Compared to sharp interface methods, diffuse interface methods are based on a different theoretical model where the interface is treated as a smooth transition region from one phase to another. The original idea can be found in [2,3], and [1] is a useful review of the diffuse interface model. In [12, 22], diffuse interface model is applied for Newtonian and non-Newtonian flows.

The main challenge for simulating the phase-field evolution is that very fine meshes are needed for resolving thin interfaces. In order to produce physically correct results one needs a very thin interface and it is almost impossible to solve the problem practically when using uniform meshes. In past years, many adaptive mesh techniques have been proposed which can be classified as adaptive mesh refinement methods and adaptive mesh redistribution methods, see [14]. In this work, we will simulate multi-phase flows using a moving mesh method (i.e., adaptive mesh redistribution method in the sense of [14]). In particular, we will use the moving mesh algorithms developed in Li et al. [10,11] which redistribute mesh nodes based on harmonic mapping. The moving mesh method based on harmonic mapping has been applied successfully to several complex problems including incompressible flow [5,6], reaction-diffusion systems [15], and dendritic growth [18,19]. The goal of the moving mesh method is to reduce the computational cost and to enhance the accuracy in resolving the diffuse interfaces.

In this work, the Oldroyd-B model for constitutive relation of viscoelastic fluids is used. The following section will briefly review the Oldroyd-B model for viscoelastic flow and the phase-field model for two-phase flows. Section 3 will describe the coupled equations and the finite-element formulation used in our computations. The moving mesh methods and the corresponding monitor function will be discussed in Section 4 and several numerical tests will be given in Section 5. The last section draws the conclusion with some discussions on the future works.

2 The Oldroyd-B model and the phase-field model

In this section, we will briefly review the governing equations for viscoelastic flows and the phase-field model for multi-phase flows.
2.1 The Oldroyd-B model for polymer fluids

The incompressible Navier-Stokes equations read

\[ \nabla \cdot \mathbf{u} = 0, \quad (2.1a) \]
\[ \rho \left( \frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} \right) = \nabla \cdot (-p \mathbf{I} + \mathbf{T}) + \rho \mathbf{g}, \quad (2.1b) \]

where \( \mathbf{u} \) is velocity, \( \rho \) is density, \( p \) is pressure, \( \mathbf{g} \) is gravity and \( \mathbf{T} \) is extra-stress tensor. In Newtonian fluids the extra-stress tensor is \( \mathbf{T} = 2\mu \mathbf{D} \), and \( \mathbf{D} = \frac{1}{2} \left[ (\nabla \mathbf{u})^T + \nabla \mathbf{u} \right] \) is the strain rate.

To study viscoelastic fluids, the Upper Convected Maxwell (UCM) model is a simple one to begin with. In this model the relation between extra-stress tensor and strain rate is given by

\[ \mathbf{T} + \lambda \nabla \mathbf{T} = 2\mu \mathbf{D}, \]

where \( \lambda \) is the relaxation time of stress, \( \mu \) is the total viscosity and

\[ \nabla \mathbf{T} = \frac{\partial \mathbf{T}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{T} - (\nabla \mathbf{u})^T \cdot \mathbf{T} - \mathbf{T} \cdot (\nabla \mathbf{u}) \]

is the upper-convected derivative of stress tensor.

In Oldroyd-B model the extra-stress tensor is divided into polymeric and Newtonian parts,

\[ \mathbf{T} = \tau + 2\mu_s \mathbf{D}, \]

where \( \mu_s \) is the Newtonian viscosity. The constitutive relation between polymeric stress and strain rate is given by

\[ \tau + \lambda \nabla \mathbf{T} = 2\mu_p \mathbf{D}, \]

where \( \mu_p = \mu - \mu_s \) is the polymeric viscosity. Now the conservation of mass, momentum with constitutive equations can be written as

\[ \nabla \cdot \mathbf{u} = 0, \quad (2.2a) \]
\[ \rho \left( \frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} \right) = \nabla \cdot (-p \mathbf{I} + \tau + 2\mu_s \mathbf{D}) + \rho \mathbf{g}, \quad (2.2b) \]
\[ \tau + \lambda \nabla \mathbf{T} = 2\mu_p \mathbf{D}. \quad (2.2c) \]

2.2 Phase-field model for interfacial dynamics

In phase-field model, the fluid phase is represented by a phase function \( \phi \). Let \( \phi = \pm 1 \) denote two stable phases and the interface is a smooth transition region where \(-1 < \phi < 1\). In order to model the interfacial dynamics, we need to define a free energy of the interface.
The Cahn-Hilliard type free energy of a phase-field is a function of the phase parameter $\phi$ defined as

$$f(\phi) = \frac{\alpha}{2} |\nabla \phi|^2 + \beta g(\phi), \quad (2.3)$$

where $\sqrt{\alpha/\beta}$ produces the surface tension and $\sqrt{\alpha/\beta}$ controls the interfacial width. The first term on the right hand side of (2.3) is the gradient energy and the second one is the bulk energy which has two minima corresponding to two stable phases. The chemical potential is the rate of change of the free energy with respect to $\phi$

$$\epsilon = \frac{\delta \int_{\Omega} f(\phi) d\Omega}{\delta \phi} = -\alpha \nabla^2 \phi + \beta g'(\phi). \quad (2.4)$$

Finally the phase parameter $\phi$ is evolved by the Cahn-Hilliard equation:

$$\frac{\partial \phi}{\partial t} = \gamma \nabla^2 \epsilon = \gamma \nabla^2 \left[ -\alpha \nabla^2 \phi + \beta g'(\phi) \right], \quad (2.5)$$

where $\gamma$ is the mobility. The interfacial width is of order

$$\epsilon \sim \sqrt{\alpha/\beta}. \quad (2.6)$$

If the phase is convected by fluid velocity and if we let

$$\alpha = k\epsilon^2, \quad \beta = k, \quad g(\phi) = \frac{1}{4}(\phi^2 - 1)^2,$$

then

$$\frac{\partial \phi}{\partial t} + \mathbf{u} \cdot \nabla \phi = \gamma \nabla^2 [k(\phi^3 - \phi - \epsilon^2 \nabla^2 \phi)], \quad (2.7)$$

which has an equilibrium profile

$$\phi(x) = \tanh\left(\frac{x}{\sqrt{2}\epsilon}\right).$$

As $\epsilon \to 0$, the ratio $a/\epsilon$ produces the interfacial tension $\sigma$ in the classical sense [9]:

$$\sigma = \frac{2\sqrt{2}}{3} k\epsilon. \quad (2.8)$$

The original fluid momentum equation (2.1b) coupled with surface tension becomes

$$\rho(\phi) \left( \frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} \right) = \nabla \cdot (-\rho \mathbf{1} + T(\phi)) + \epsilon \nabla \phi + \rho(\phi) g, \quad (2.9)$$

where the density $\rho$ and the stress tensor $T$ depend on the phase parameter $\phi$. 

3 The coupled system

For two-phase non-Newtonian and Newtonian mixture, the incompressible Navier-Stokes equations with the Oldroyd-B model are

\begin{align}
\nabla \cdot \mathbf{u} &= 0, \\
\rho(\phi)\left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u}\right) &= \nabla \cdot \left(-p \mathbf{I} + \mathbf{T}(\phi)\right) + \rho(\phi) \mathbf{g}, \tag{3.1a}
\end{align}

\begin{align}
\tau + \lambda \left[\frac{\partial \tau}{\partial t} + \mathbf{u} \cdot \nabla \tau - (\nabla \mathbf{u})^T \cdot \tau - \tau \cdot (\nabla \mathbf{u})\right] &= 2\mu_p \mathbf{D}, \tag{3.1b}
\end{align}

\begin{align}
\frac{\partial \phi}{\partial t} + \mathbf{u} \cdot \nabla \phi &= \gamma \nabla^2 [k(\phi^3 - \phi - \epsilon^2 \nabla^2 \phi)], \tag{3.1c}
\end{align}

where \( \mathbf{c} = k(\phi^3 - \phi - \epsilon^2 \nabla^2 \phi) \) is the chemical potential, \( k \) is a constant determined by (2.8) and \( \gamma \) is the mobility of \( c \). In (3.1a), the density and stress tensors, which depend on the phase function \( \phi \) are given by

\begin{align}
\rho(\phi) &= \frac{1}{2} \rho_1 + \frac{1}{2} \rho_2, \\
\mathbf{T}(\phi) &= \frac{1}{2} \left(\tau + 2\mu_1 \mathbf{D}\right) + \frac{1}{2} \left(2\mu_2 \mathbf{D}\right). \tag{3.2b}
\end{align}

If both components are non-Newtonian, the formulations are similar. Moreover, appropriate boundary conditions should be chosen for specifically given problems.

Although there are many ways to discretize the governing equations above, we seek for ones that are easy to implement. One thing in mind is to make the resulting matrices symmetric positive definite so that we can make use of the Gauss-Seidel-iteration based algebraic multi-grid solver whose average time complexity is \( O(N) \). Let \( \hat{\mathbf{u}}, \hat{\mathbf{p}}, \hat{\tau} \) and \( \hat{\phi} \) be the test functions corresponding to \( \mathbf{u}, \mathbf{p}, \tau \) and \( \phi \), respectively. For the momentum equations we adopt a simple pressure correction projection scheme to satisfy the incompressibility condition (3.1a). The time discretization form reads

\begin{align}
\int_\Omega \left\{ \rho(\phi_{n+1}) \left(\frac{\mathbf{u} - \mathbf{u}_n}{\Delta t} + \mathbf{u}_{n+1} \cdot \nabla \mathbf{u}_{n+1} - \mathbf{g}\right) - \mathbf{c}_{n+1} \nabla \phi_{n+1}\right\} \cdot \hat{\mathbf{u}} \, d\Omega &= 0, \tag{3.3}
\end{align}

\begin{align}
\int_\Omega \nabla^2 \hat{\mathbf{p}} \, d\Omega &= \int_\Omega \nabla \cdot \hat{\mathbf{u}} \, d\Omega, \tag{3.4}
\end{align}

\begin{align}
\int_\Omega \mathbf{u}_{n+1} \cdot \hat{\mathbf{u}} \, d\Omega &= \int_\Omega \left(\mathbf{u} - \nabla \hat{\mathbf{p}}\right) \cdot \hat{\mathbf{u}} \, d\Omega. \tag{3.5}
\end{align}

In (3.3), the inner product between two vectors \( \mathbf{A} \) and \( \mathbf{B} \) is denoted by \( \mathbf{A} : \mathbf{B} \), and is given by

\[ \mathbf{A} : \mathbf{B} = \text{trace}(\mathbf{B}^T \cdot \mathbf{A}). \]
In (3.5), \( \tilde{u} \) is the intermediate velocity which is not divergence free. We compute
\[
\tilde{p} = \frac{\Delta t}{\rho(\phi_{n+1})} p
\]
instead of \( p \) to avoid extra errors due to the varying time step \( \Delta t \) and different density values. A detailed overview of projection schemes and their error estimates can be found in [8]. For constitutive equations we use the following weak formulation
\[
\int_{\Omega} \left\{ \tau + \lambda \nabla - 2\mu pD \right\} : (\hat{\tau} + \alpha \hat{u} \cdot \nabla \hat{\tau}) d\Omega = 0. \tag{3.6}
\]
Note that we choose a different test space to improve the computation; which is as same as [22]. The Cahn-Hilliard equation is solved in two steps because \( C^0 \) elements are used in our computation. The time discretization reads
\[
\int_{\Omega} \left\{ \frac{\phi_{n+1} - \phi_n}{\Delta t} + u_{n+1} \cdot \nabla \phi_{n+1} - \gamma \nabla^2 c_{n+1} \right\} \phi d\Omega = 0, \tag{3.7}
\]
\[
\int_{\Omega} c_{n+1} \phi d\Omega = \int_{\Omega} k(\phi_{n+1}^3 - \phi_{n+1} - \epsilon^2 \nabla^2 \phi_{n+1}) \phi d\Omega. \tag{3.8}
\]
Finally, the whole system can be written in the following general form
\[
\frac{U_{n+1} - U_n}{\Delta t} + F(U_{n+1}) = 0, \tag{3.9}
\]
where \( U_n \) and \( U_{n+1} \) are the solution vector at step \( n \) and step \( n+1 \) levels respectively. Using standard iteration procedure, we let \( U_{n+1}^{(0)} = U_n \) and solve
\[
\frac{U_{n+1}^{(k+1)} - U_n}{\Delta t} + F(U_{n+1}^{(k)}) = 0, \tag{3.10}
\]
repeatedly. In each iteration step, we pass the temporary solution \( U_{n+1}^{(k)} \) to \( F \) in Eq. (3.10) and obtain the updated solution \( U_{n+1}^{(k+1)} \). These sub-steps are explicit except that all the terms involving the Laplacian operators are treated implicitly. Note that the whole system is implicit; so the block Gauss-Seidel iterations are used instead of solving a large linear system directly. The system is divided into six sub-systems and the algebraic multigrid solver is applied to each of the sub-matrix blocks. In our computations, five to eight block-iterations are needed for each time step, which makes the iteration errors drop to below \( 10^{-10} \).

4 Space and time adaptation

4.1 Moving mesh method for capturing interface

Solving the Cahn-Hilliard equation requires high mesh resolution around sharp interface, usually we need two or more elements within the interfacial region. But using a uniform
fine mesh will slow down the computation significantly. In addition, if we increase the mesh resolution globally without using local refinements, it is likely to cause instability. One way to handle this is to use coarse meshes with local refinement which can reduce computational costs [22]. Because of mesh structure modification, this method needs storage reallocation which is a fairly complex process. An alternative simpler approach is to use moving mesh methods which can concentrate the mesh nodes on sharp interface without changing mesh structure and the mesh size changes smoothly near the interfacial regions. In our computations, a moving mesh strategy based on the framework of Li et al. [11] is adopted. In [11], a mesh generation technique using harmonic mappings was developed. Apart from clustering more grid points near the interfaces, another benefit we wish to gain from using the moving mesh technique is the anisotropic mesh adaptation. It is hoped that more mesh nodes should be placed along the interface normal than that on the interface tangent.

The basic idea is to redistribute mesh nodes according to the regularity of solutions, which can be achieved by using harmonic mapping. Let \( x \) and \( \xi \) be physical and logical mesh coordinates, respectively. The following Euler-Lagrange equation

\[
\frac{\partial}{\partial x^i} \left( G^{ij} \frac{\partial^2 \xi^k}{\partial x^j} \right) = 0 \tag{4.1}
\]

gives a continuous and one-to-one mapping which is differentiable and has a non-zero Jacobian, where the inverse of \( G = (G^{ij}) \) is called monitor function which is in general dependent on the underlying solutions. The corresponding mesh energy is

\[
E(\xi) = \sum_k \int_\Omega G^{ij} \frac{\partial^2 \xi^k}{\partial x^j} \frac{\partial^2 \xi^k}{\partial x^i}. \tag{4.2}
\]

To make the moving mesh simulations effective, the key is to develop an efficient mesh redistribution procedure so that this part will cost as little as possible comparing with the solution evolution part. This issue was discussed in a recent paper of Di et al. [4].

In order to increase mesh resolution at interface we use the following function to realize the transition zone of the phase function:

\[
g(x) = |\nabla \phi(x)|^2. \tag{4.3}
\]

Consequently, the corresponding monitor function is

\[
m(x) = \sqrt{1 + \alpha_1 g_s(x)}, \tag{4.4}
\]

where \( g_s \) is a smoothed version of \( g \) and \( \alpha_1 \) is a scaling parameter. It is pointed out that the smoothed function \( g_s \) is still of compact support, which is non-zero only within or near the interface because the phase function is constant away from the interface. This will make the mesh near the interface badly stretched. An example is shown in Fig. 1 which
comes from our first numerical example in Section 5. However, we still need regular
elements with enough resolution for correctly computing the velocity and viscoelastic
stress field away from the interface. One remedy is to smooth the function several times
but this approach may be too costly. Instead, we solve a heat equation implicitly to diffuse
the monitor function which is a much faster process due to the use of the algebraic multi-
grid solvers. Let $g_d(x) = g(x,1)$, where $g(x,t)$ satisfies

$$
\frac{\partial g}{\partial t} = \nu \nabla^2 g.
$$

(4.5)

We define the new monitor function as

$$
m(x) = \sqrt{1 + \alpha_1 g_s(x) + \alpha_2 g_d(x)},
$$

(4.6)

where $\alpha_2 > 0$ is a blend factor. This monitor function contains both $g_s$ and $g_d$. On the
one hand, we need the original high frequency monitor to control the local element size
at the interface. On the other hand, using only $g_d$ we may not have sufficiently small
elements at the interface since $g_d$ is very smooth. A similar idea was also used in [18].
The diffused monitor function with a few smooth steps produces more reasonable mesh
distribution as demonstrated in Fig. 1. It is seen from Fig. 1 that the monitor (4.6) yields
smaller ratio between the large and small element sizes which is found important for
solving the multi-phase flow problems.

### 4.2 Adaptive time step

In moving mesh implementation, another important consideration is about adaptive time
step due to the varying element size and velocity field. Time step must be determined
based on physical and artificial parameters to avoid inaccuracy or instability so that we
do not need to set time step manually. From the coupled equations described in Section
3, we consider the terms involving spatial derivatives. Roughly speaking, the following
condition should be considered

$$
\Delta t < \min \left\{ \Delta t_{\text{max}}, \frac{h}{|u|}, \frac{C_1 h^4}{\gamma \sigma \epsilon} \right\},
$$

(4.7)

where $C_1$ and $C_2$ are constants, $h$ is the local element size and $u$ is the local velocity norm.
It is obvious that the time step is constrained by the CFL condition and the Cahn-Hilliard
equation. For correct phase-field computation, $h < \epsilon$ should be satisfied around interfacial
regions and increasing $\epsilon / h_{\text{min}}$ may produce more accurate results while slow down the
overall computation. So we keep $\epsilon / h_{\text{min}}$ being some constant to balance the accuracy and
efficiency. A more detailed analysis of $\epsilon$ and $h$ can be found in [9].

In our numerical computations, we choose $\Delta t_{\text{max}} = 0.01$ in (4.7) in all of our numerical
experiments.
5 Numerical results

5.1 Vibrating of square bubbles

In our first experiment we consider the surface tension effects of a bubble by measuring its shape deformation history. A similar test of Newtonian flow is investigated in [12] without quantitative measurement of the bubble shape. At first a square bubble is placed in the center of domain shown as in Fig. 1 and the bubble begins to vibrate and its shape finally becomes round. The domain size is $1 \times 1$ and the bubble size is $0.2 \times 0.2$. Both fluid components are non-Newtonian and the physical parameters are $\epsilon = 0.01, 0.005, 0.0025$, $\mu_s = \mu_p = \mu_{s2} = \mu_{p2} = 0.1$, $\lambda = 0.1$, $\gamma = 10^{-4}$ and $k = 0.1$.

We use both moving mesh and uniform mesh to validate the convergence. The shape
deformation history

\[ D = \frac{L - S}{L + S} \]  \hspace{1cm} (5.1)

is plotted in Fig. 2, where \( L \) and \( S \) are the largest and smallest distance from bubble center to the interface. It is observed from Fig. 2 that the 32 \times 32 and 50 \times 50 moving mesh results and the 100 \times 100 uniform mesh results are in good agreement, while 50 \times 50 and 64 \times 64 uniform meshes produce poor results. A performance statistics for this example is given
Table 1: Example 5.1: performance statistics.

<table>
<thead>
<tr>
<th>Grid Size</th>
<th>Mesh nodes</th>
<th>Adaptive</th>
<th>CPU (sec.)</th>
</tr>
</thead>
<tbody>
<tr>
<td>32 × 32</td>
<td>1259</td>
<td>Yes</td>
<td>7614</td>
</tr>
<tr>
<td>40 × 40</td>
<td>1932</td>
<td>Yes</td>
<td>11765</td>
</tr>
<tr>
<td>50 × 50</td>
<td>3009</td>
<td>Yes</td>
<td>18394</td>
</tr>
<tr>
<td>50 × 50</td>
<td>3009</td>
<td>No</td>
<td>12073</td>
</tr>
<tr>
<td>64 × 64</td>
<td>4880</td>
<td>No</td>
<td>20132</td>
</tr>
<tr>
<td>80 × 80</td>
<td>7579</td>
<td>No</td>
<td>32840</td>
</tr>
<tr>
<td>100 × 100</td>
<td>11833</td>
<td>No</td>
<td>51552</td>
</tr>
</tbody>
</table>

Table 2: Example 5.3: test results of rising bubbles.

<table>
<thead>
<tr>
<th>Case</th>
<th>σ</th>
<th>μ</th>
<th>Newtonian matrix</th>
<th>U</th>
</tr>
</thead>
<tbody>
<tr>
<td>A</td>
<td>0.1</td>
<td>0.562</td>
<td>Yes</td>
<td>0.090</td>
</tr>
<tr>
<td>B</td>
<td>0.1</td>
<td>0.562</td>
<td>No</td>
<td>0.086</td>
</tr>
<tr>
<td>C</td>
<td>0.1</td>
<td>0.0562</td>
<td>Yes</td>
<td>0.554</td>
</tr>
<tr>
<td>D</td>
<td>0.01</td>
<td>0.187</td>
<td>Yes</td>
<td>0.252</td>
</tr>
<tr>
<td>E</td>
<td>0.01</td>
<td>0.562</td>
<td>No</td>
<td>0.086</td>
</tr>
</tbody>
</table>

in Table 1, from which it is observed that a considerable CPU time is saved.

To reduce the area change ratio, defined by \( \frac{(\text{Area}(t) - \text{Area}(0))}{\text{Area}(0)} \), smaller \( \epsilon \) may be chosen. Fig. 3 compares the area change ratio with different interfacial widths. It is observed from the figure that smaller interfacial widths preserves the area better.

5.2 Deformation of bubbles in shear flow

5.2.1 Deformation of Newtonian bubble

The deformation of a bubble in shear flow is also studied to validate our algorithm. The first test is Newtonian bubble in Newtonian matrix which is the same as that in [22] except the Reynolds number \( Re = 1 \) is used in our computation because the inertia in our computation is not ignored. The capillary number \( Ca = \mu a / \sigma = 0.1 \) is used, where \( a \) is drop radius and \( \kappa \) is shear rate. We set \( \mu = \mu_1 = \mu_2, \gamma = 10^{-4} \) and \( \epsilon = 0.01 \). The domain size is \( 16a \times 8a \) where the bubble is placed in the center of domain.

Again the shape deformation \( D = (L - S) / (L + S) \) is considered, which is plotted in Fig. 4 where \( L \) and \( S \) are the longest and shortest distance from bubble center to interface, respectively. The deformation is also reported in [22] at \( t = 4 \) with several small values of \( \epsilon \) (\( \epsilon = 0.01,0.005 \) and 0.0025 vs. our value \( \epsilon = 0.01 \)). It is shown that our results using moving mesh methods is in good agreement with those given in [22]. At \( t = 4 \) we obtained \( D = 0.1081 \) which is close to the values of \( D = 0.1067,0.1085,0.1087 \) (corresponds to \( \epsilon = 0.01,0.005 \) and 0.0025 respectively) given in [22].

Fig. 5 plots the time-step history. When velocity is small, the time-step is almost constant since solving the Cahn-Hilliard equation requires small time-steps.
5.2.2 Deformation of viscoelastic bubble

The next test is a non-Newtonian bubble in a Newtonian matrix. We set $\gamma = 0.0016$ and $\epsilon = 0.04$ as in [21]. In this test we compared the shape deformation using different Deborah numbers $De = \lambda \nu$. Fig. 6 plots the deformation $D = (L-S)/(L+S)$ for $De = 0.5, 1, 2$. It is observed that the non-Newtonian bubbles overshoot in $D$. The overshooting phenomenon agrees with the observations in [21] except that the curves of non-Newtonian bubbles are all above the Newtonian curve. This may be caused by the inertia of the bub-
Figure 6: Example 5.2.2: deformation of viscoelastic bubbles in Newtonian matrix with different Deborah numbers (3841 mesh nodes are used with \( \epsilon = 0.04 \))

Figure 6 shows the deformation of viscoelastic bubbles in Newtonian matrix with different Deborah numbers. The non-Newtonian bubbles have smaller viscosity and move faster than the Newtonian bubble at the beginning; and the polymeric stress is damped due to the use of the coarse meshes.

5.3 Rising of bubbles

We also compute the rising of Newtonian bubbles in Newtonian and non-Newtonian fluids. The configuration is same as that in [22]: a bubble with diameter \( d = 1 \) is placed at \((0,2)\) in a \(4 \times 15\) domain and we use a mesh with 6134 nodes for half of the domain. The fixed parameters are \( \rho_1 = 0.5, \rho_2 = 1, g = 2 \) and the viscosity ratio \( \mu_1 / \mu_2 = 1 \). If the matrix is non-Newtonian, we also set \( \mu_p = \mu_s = 0.5 \mu_2 \). We measured the steady state velocity \( U \) of the rising bubbles and the results are listed in Table 2 where the parameters for 5 different cases are also listed.

Fig. 7 presents the corresponding bubble shapes for the tests (a)-(e) listed in Table 2, which correspond to \( t = 89.7s, 98.2s, 16.6s, 34.7s \) and \( 101.3s \), respectively. It is observed that our moving mesh results are in good agreement with the cases a/b/d and the non-Newtonian case in [22]. The slight difference may be caused by the element size since the moving mesh method cannot control the smallest element size precisely.

It is noticed that both Cases (b) and (d) are non-Newtonian with the same \( \mu_1 \), but the surface tension is different: the surface in Case (e) is 10 times smaller that in Case (b). Consequently, it is observed that the trailing edge dragged by the viscoelastic stress at the bottom of bubble is much longer in Case (e).

Finally, we show the mesh distribution around the interface in Fig. 8. It is noticed that the element sizes away from the interface are not too small. This is in fact quite impor-
Figure 7: Example 5.3: final shape of the rising bubble: (a) in Newtonian and Newtonian fluids with the surface tension $\sigma = 0.1$ and viscosity $\mu_1 = 0.562$; (b) in Newtonian and non-Newtonian fluids, with the surface tension $\sigma = 0.1$ and viscosity $\mu_1 = 0.562$; (c) in Newtonian and Newtonian fluids with the surface tension $\sigma = 0.1$ and viscosity $\mu_1 = 0.0562$; (d) in Newtonian and Newtonian fluids with the surface tension $\sigma = 0.01$ and viscosity $\mu_1 = 0.187$; and (e) in Newtonian and non-Newtonian fluids, with the surface tension $\sigma = 0.01$ and viscosity $\mu_1 = 0.562$. 
tant in obtaining accurate numerical approximations. In fact, we are not only interested in resolving the phase function and its interface, but also in obtaining accurate flow fields which are governed by the nonlinear system (3.1a). As the moving mesh methods correspond to a continuous process, a reasonable resolution for the flow parameters in the whole solution domain is required. In our numerical computations, it is found that if the element sizes are too big even quite far away from the interface then the rising velocity and viscoelastic stress will be underestimated. In other words, if the flow fields are not well resolved then some physical details may be missing, which may have direct effect on the fluid velocity and the bubble shape.

6 Conclusion

In this work, we successfully applied the moving finite-element method for two-phase viscoelastic flow simulations. Since the mesh redistribution procedure normally requires to solve large size matrix equations, we decoupled the matrix equations to a much simpler block-tridiagonal type which can be efficiently solved by a particularly designed multi-grid method which is described in [10]. It is found that the moving mesh approach can lead to considerable savings in grid numbers and also the CPU time.

In our computations, it is observed that the balance between the largest and smallest element sizes should be given. In other words, we need to diffuse the monitor functions so that a certain portion of mesh nodes are distributed away from the interfaces. This will produce reasonably accurate velocity fields that in turn gives more accurate approximation to the phase functions.

In future works, we will study how the moving mesh interpolation affects the solution
of velocity field for unsteady problems. We will also extend the present 2D algorithms to 3D simulations.

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